- **MATERIAL: Base & Stud –** Steel, Zinc Plated
  Isolator Fig. 1 – Natural Rubber / Styrene-Butadiene
  Fig. 2 – Nitrile Rubber Compound

- **FOR LOADS UP TO 1632 kgf (3598 lb.)**

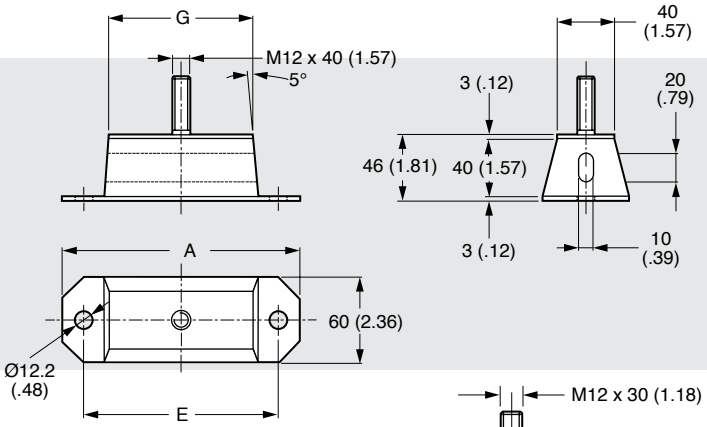
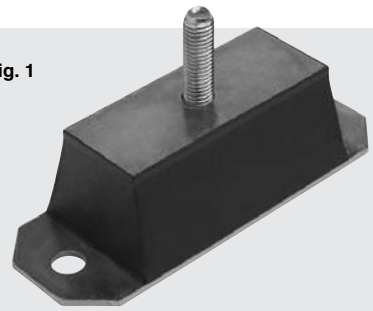


Fig. 1



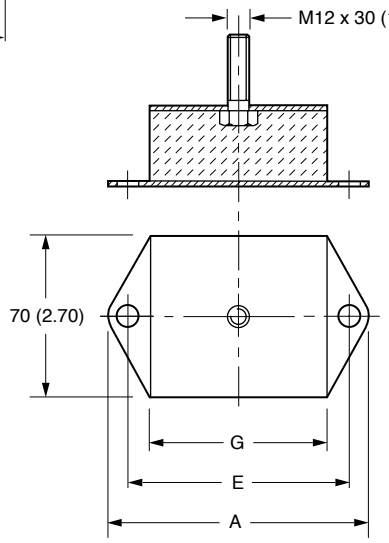


Fig. 2

Metric

2 Base Mounts

Catalog Number	Fig. No.	E	A	G	Max. Load kgf (lb.)	Durometer ± 5
V12Z06MA1156005	1	85 (3.35)	115 (4.53)	50 (1.97)	510 (1124)	45
V12Z06MA1656012		135 (5.32)	165 (6.5)	100 (3.94)	1224 (2698)	
V12Z06MA2156014		185 (7.28)	215 (8.47)	150 (5.91)	1428 (3148)	
V12Z06MA2656016		235 (9.25)	265 (10.43)	200 (7.87)	1632 (3598)	
V12Z06MB1307007	2	105 (4.1)	130 (5.12)	80 (3.15)	663 (1462)	45
V12Z06MB1307010		105 (4.1)	130 (5.12)	80 (3.15)	969 (2136)	60

NOTE: Dimensions in ( ) are inch.

1. Determine the load that each mount will bear when supporting the equipment weight. Total weight divided by the number of mounting positions is the load for each mount. This is only true when having even weight distribution. Otherwise, distribute weight accordingly.
2. Determine the lowest forcing frequency of the vibration source to be supported by the mounts. This is usually equal to the operating speed in revolutions per minute.
3. Choose the percent isolation that will be satisfactory for the purpose. Except for special cases, 81% isolation is generally considered satisfactory.
4. Referring to the Basic Vibration Chart below, find the static deflection for the forcing frequency (Step 2, above) at the chosen percent isolation (Step 3). Note that a mount must give at least this minimum static deflection, with the specific load applied,to provide the desired isolation.
5. Select the mount series with the physical features (shape, attachment facilities,"fail-safe" safety feature, load range, etc.) required by the application.
6. a) Having selected the mount series, refer to the in dividual styles, and note the styles whose maximum loads are greater than the load each mount is to carry.

   b) Referring to the load deflection graphs of the styles likely to be chosen, locate the applied load value (Step 1, above) on the appropriate graph; i.e., compression and/or shear.

   c) Moving horizontally to the right on the graph, locate the point of intersection with the minimum static deflection found in step 4.

   d) Mounts with curves above this point of intersection cannot be used, as the load (Step 1) is not sufficient to produce the required minimum deflection (Step 4).

   e) Mounts with curves below the point of intersection can be used as, at the given load, the deflection will be greater than the minimum required. Note, however, that if the applied load is above the line x--x on a curve, the mount is not recommended for this static load.

   f) More than one style may have load-deflection curvesthat are suitable. The final selection can depend on other requirements such as the cost of the mounts, possible in-creased load requirements in the future, relative advantage of additional isolation, space available for the mounts, constraints on allowable deflection, attachment requirements, etc. However, in the absence of any overriding consideration, usually the mount that is selected has its curve closest to the point of intersection (Step 6c); i.e., the mount with the minimum deflection at the applied load.
7. Select the mount that is designed to operate in your temperature range and environment.

Vibration Frequency vs Static Deflection vs Isolation Efficiency